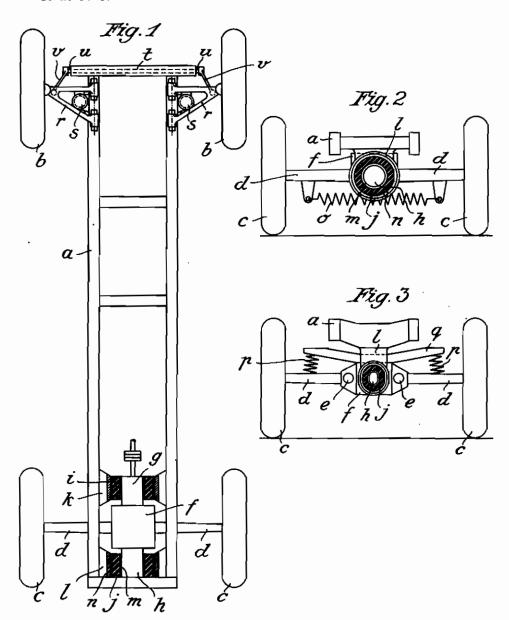
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O. WINKELMANN
VEHICLES, PARTICULARLY MOTOR VEHICLES
HAVING OSCILLATING AXLES
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BY A. P. C.



INVENTOR: OTTO WINKELMANN

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VEHICLES, PARTICULARLY MOTOR VEHI-CLES HAVING OSCILLATING HALF AXLES

Otto Winkelmann, Stuttgart-Degerloch, Germany; vested in the Alien Property Custodian

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The present invention relates to vehicles, more particularly to motor vehicles having oscillating or pendulum half axles. Motor vehicles of this kind often tend to oscillate about the frame or the car body, i. e. to perform the so-called rocking which may hardly be obviated by larger dimensioning the oscillating members. This latter method, besides causing an undesired increase in the weight of the vehicle and a too rigid and torsional stiff frame, has the drawback of an undamped transmission of shocks and noises.

Now, the invention provides an increase of the torsional elasticity between the axles of the vehicle to reduce the natural vibrations of the car system about its longitudinal axis, so that the vibrating movements are no longer experienced by the passengers to be troublesome, but appear in the form of weak slowly occurring amplitudes. To this purpose the axle aggregate, consisting more particularly of pendulum half axles, is connected to the rest of the vehicle by the interposition of elastic members, for instance rubber buffers, in a manner to be elastically rotatable about a longitudinal axis, the springs, serving the purpose of shock absorption of the wheels, and the rubber buffers being connected one after the other (therefore in series) in the sense of the power transmission from the wheels to the car body. Hereby simultaneously the transmission of shocks and noises from the wheels to the 30 frame or the car body is prevented.

A further reduction of the oscillating phenomenons may be obtained by arranging the springs, absorbing the shocks of the wheels, in springs, i. e. totally or partially give way due to oppositely directed or unequal large lifting movements of the wheels of a wheel axle without exerting upon the car body or the supporting member carrying the members guiding the wheel a 40 tically arranged at the frame and a front wheel force returning these elements (direction force). The wheels for this purpose are thus arranged relatively to each other that they mutually absorb their shocks particularly directly.

By the torsional elastic connection of the axle 45 aggregate to the rest of the vehicle, however, the lifting action exerted upon the car body on lateral inclinations of the latter and thereby the safety of the car body when taking curves is reduced, more particularly if compensating springs are provided at this axle. To increase this safety in taking curves, however, the other axle preferably may be provided with an additional spring arrangement, for instance a so-called balancer, effecting setting upright the car body.

If it is, however, desirable to reduce only relatively slightly the lifting action exerted upon the car body by the axle torsionally elastically connected to the rest of the vehicle, or, if the torsional elasticity of the connection is sufficient to remove the undesired oscillating phenomenons, the springs, absorbing the shocks of the wheels, may also bear against the supporting member, for instance a differential gear, which carries the pendulum half axles and torsionally elastically is connected to the rest of the vehicle. In this case, on lateral inclinations of the car body and thereby of the supporting member also with an amplitude admitted by the torsional elasticity provided between the supporting member and the rest of the vehicle, the springs absorbing the shocks of the wheels also act in a setting upright manner upon the car body.

To effect the torsional elastic connection of the supporting member, for instance the differential gear casing, carrying the pendulum half axles or the like, to the rest of the vehicle, preferably rubber rings of annular, oval or similar form or pneumatic devices, e. g. elastic, air filled containers of rubber or elastic metal or other elastic members constructed and arranged in such a manner are provided which particularly with the elements to be united together, are adhesively connected and effect a resiliency essentially or exclusively in the direction of rotation about a longitudinal axis, these resilient or elastic members preferably being arranged in front and in rear of the oscillating or pendulum half axles.

In the accompanying drawing some construcsuch a manner that they act as compensating 35 tions according to the invention are shown diagrammatically and by way of example.

In this drawing:

Fig. 1 is a plan view of a vehicle frame provided with a rear axle aggregate torsionally classuspension employing an additional spring arrangement for setting upright the carriage body.

Fig. 2 is a rear elevation of the rear axle aggregate according to Fig. 1, and

Fig. 3 is a rear view of a modification of the rear axle aggregate of the construction shown in

Connected to the frame a are the front wheels b and the rear wheels c which latter are mounted upon oscillating or pendulum half axles d. These half axles are linked to the vehicle in the middle longitudinal plane of same, or, as shown for instance in Fig. 3, by means of lateral links e to the differential gear casing f. The latter is pro-55 yided with projections g and h directed forward2 414,042

ly and rearwardly respectively which extend in the longitudinal direction of the vehicle and arc surrounded by rubber rings i and j respectively inserted in corresponding bearing eyes k and l respectively of the frame. The rubber rings preferably are adhesively connected for instance by vulcanization to inner and outer metal sleeves m and n respectively. The inner metal sleeves m are fixed to the projections g and h respectively, the outer metal sleeves n to the bearing 10 cyes k and l respectively of the frame.

In the construction of the rear axle according to Fig. 2, the rubber rings have circular cross section. To absorb the shocks of the wheels one or more tension springs o are provided the two 15 ends of which are connected to projections of the half axles d. The springs o are so arranged that the two half axles d mutually absorb the shocks acting upon the wheels supported by these half axles. In a manner known per se a so-called 20 copensating action of the springs, absorbing the shocks of the wheels, is obtained hereby.

According to the construction of the rear axle shown in Fig. 3 the springs, absorbing the shocks of the wheels or the half axles respectively, and 25 constructed for instance as non-guided helical springs bear with their upper ends against a beam q fixed to the differential casing f and, therefore, together with the latter is mounted by means of the rubber rings i and j in a torsional 30 elastic manner with regard to the rest of the vehicle. In the present case the rubber rings have an oval cross section so as to effect a progressively increasing elastic resistance when the differential gear casing, serving as supporting 35 member for the oscillating half axles, yields with regard to the frame. A particularly large initial resiliency may thereby be chosen without the total resiliency assuming inadmissible large values.

The front wheels b may for instance, in a man- 40 ner known per se, be suspended from the frame a by two links r arranged one above the other in the manner of link parallelograms and the shocks of these wheels may be absorbed by a helical spring s the lower end of which bears against the 45 lower link and the upper end of which bears against a bearing block fixed to the frame. To increase the safety of the frame when taking curves, a torsional rod-like spring u is, moreover, rotatably mounted in a tubular cross beam t of 50 the frame upon the ends of which spring levers vare fixed the free ends of which may in any suitable manner be connected to the wheel suspension, for instance to the wheel support or one of the guide links. This torsional rod-like spring 55 u is deformed only, if unequal lifting movements of the front wheels occur.

Owing to the torsional elastic connection of the rear axle aggregate to the frame or to the car body respectively by means of the rubber rings i and j unequally directed or unequal large lifting movements of the rear wheels may partially and mutually be compensated without being transmitted to the frame or to the car body respectively. Simultaneously the number of oscillations of the total oscillating system, formed

by the frame and the car body as well as by the two axles, about the longitudinal axis of the vehicle is reduced. The absorption of shocks of the rear wheels is weaker, particularly in connection with the construction according to Fig. 2 and shocks and noises are dampened in a most agreeable manner due to metallic contact between axle- and frame- members or members of the car body respectively being obviated. Simultaneously, however, the torsional rod-like spring u, i. e. a so-called balancer, acting as an additional spring for lifting the car body eventually effects a sufficient safety of the frame or the car body in taking curves.

Of course, the invention is not to be limited to the constructional examples shown. For instance, the individual features of the constructions shown in Figs. 2 and 3 could be exchanged one against the other by using for instance in the construction according to Fig. 2 ovally constructed rubber rings or by employing equalizing springs 0 in the modification shown in Fig. 3 instead of the springs p or additionally thereto. Also instead of rubber rings individual rubber plates or the like arranged in a circle or in a similar manner may be provided.

Instead of the rear axle or simultaneously therewith, moreover, the front axle also may be torsionally elastically connected to the rest of the vehicle by elastic members. Moreover, the elastic members need not be arranged directly at the axle, but may be interposed at other places also, as long as they are arranged in the power transmission path between the two axles or between the one axle and the car body respectively. A special frame may be omitted for instance by the use of self-supporting car bodies.

The torsional elastic connection of the axle aggregate to the rest of the vehicle is of special importance when using gauge altering pendulum half axles. The torsional elastic connection, however, may be employed also in connection with other arrangements of oscillating axles or independent wheel suspensions and eventually in connection with rigid axles too. It may be furthermore used in connection with vehicles generally and also to vehicles (motor vehicles) with more than two axles.

The invention may, moreover, also be used without a balancer provided at the axle arranged opposite the torsionally elastically mounted axle. The wheels of the last mentioned axle, i. e. particularly the wheels of the front axle, may eventually be suspended in any desired manner from the frame or the car body of the vehicle respectively. For these wheels, however, principally parallel or link parallelogram connections come into consideration.

Generally speaking it is of course also possible to apply the invention only to one or some of the axles viz. axle-pairs of one vehicle and it is furthermore possible to provide a vehicle with one or some oscillating axles or pairs of oscillating half axles only.

OTTO WINKELMANN.